

# 000 001 002 003 004 005 006 007 008 009 010 011 012 013 014 015 016 017 018 019 020 021 022 023 024 025 026 027 028 029 030 031 032 033 034 035 036 037 038 039 040 041 042 043 044 045 046 047 048 049 050 051 052 053 ETGS: EXPLICIT THERMODYNAMICS GAUSSIAN SPLATTING FOR DYNAMIC THERMAL RECONSTRUCTION

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Paper under double-blind review

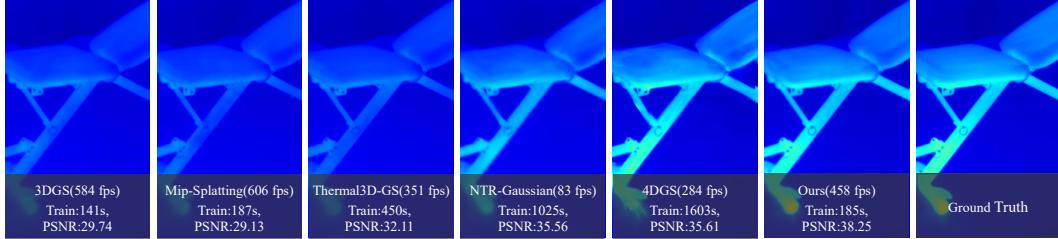


Figure 1: Our method achieves high-quality rendering of dynamic thermal scenes with efficiency comparable to static methods (Kerbl et al. (2023); Yu et al. (2024)). The key to this performance is the novel explicit modeling of dynamic thermal Gaussians based on thermodynamics, which significantly speeds up scene optimization and synthesis of new views, while achieving state-of-the-art quality.

## ABSTRACT

We propose ETGS, a method for reconstructing dynamic thermal scenes by embedding explicit thermodynamic modeling into 3D Gaussian Splatting. Each Gaussian is equipped with physically interpretable thermal parameters, and its thermodynamics evolution is described by a first-order heat-transfer ODE with an analytical closed-form solution. This formulation avoids numerical integration, enables efficient rendering at arbitrary timestamps, and naturally handles irregular sampling and out-of-order observations. We also introduce the Rapid Heat Dynamics (RHD) dataset, which provides millisecond-aligned RGB-IR image pairs covering typical thermal processes such as cooling, warming, heating, and heat transfer. Experiments on RHD show that ETGS captures rapid thermal dynamics more accurately than existing static and dynamic baselines, while maintaining training and rendering efficiency close to that of static 3DGS. Code and dataset will be released.

## 1 INTRODUCTION

Thermal imaging is a non-contact temperature measurement method that can directly capture thermal radiation signals from an object’s surface and maintains stable operation even in complex environmental conditions such as low light levels (Wilson et al. (2023); Li et al. (2024a); Xu et al. (2025); Shin & Park (2025)). Compared to traditional visible light imaging, thermal imaging not only provides geometric structural information but also reflects the temperature distribution characteristics of an object, thus demonstrating unique advantages in 3D scene modeling and physical process analysis (Aibibu et al. (2025); Nowakowski & Kaczmarek (2025); Wang et al. (2025); Wu et al. (2020); Ramon et al. (2022)). Therefore, combining thermal imaging with 3D reconstruction to generate temperature scene models that evolve over time has become a research hotspot in recent years.

Neural Radiance Fields (NeRF) (Mildenhall et al. (2021)) uses implicit neural networks to model scenes, producing high-quality results in novel perspective synthesis tasks. In thermal scene modeling, Thermal-NeRF (Ye et al. (2024)) and ThermoNeRF (Hassan et al. (2024)) extend NeRF to

IR images, achieving 3D temperature field reconstruction. However, NeRF’s implicit representation suffers from geometric instability and inefficiencies in training and rendering (Wang et al. (2024); Li et al. (2023; 2024b); Korhonen et al. (2024)), making it difficult to implement in dynamic scenes. Unlike NeRF, 3D Gaussian Splatting (3DGS) (Kerbl et al. (2023)) uses explicit Gaussians to model scenes, and all parameters (position, size, orientation, color, etc.) can be learned, resulting in strong scalability and downstream compatibility (Fei et al. (2024); Charatan et al. (2024); Tang & Cham (2024)). Recently, methods such as Thermal3D-GS (Chen et al. (2024)) and TGA-GS (Zou et al. (2025)) have introduced 3DGS into thermal reconstruction tasks, achieving 3D modeling and perspective synthesis of thermal scenes. However, they are all limited to static scenarios and fail to capture the dynamic process of temperature change over time, so they have obvious limitations in thermodynamic analysis.

To address dynamics, some work has introduced temporal modeling on 3DGS. 4DGS (Wu et al. (2024)) incorporates temporal variables into the Gaussian representation, enabling it to reconstruct dynamic appearances, but does not consider thermophysical processes. ThermalGS (Liu et al. (2025)) attempts to drive the temporal evolution of the Gaussian through temporal embedding and semantic information, but is essentially still data-driven modeling and lacks thermodynamic consistency. NTR-Gaussian (Yang et al. (2025)) introduces thermodynamic equations into the Gaussian framework, but it relies on implicit neural networks and integral inference, which limits its training and rendering efficiency.

To address these limitations, we propose ETGS, a reconstruction method for dynamic thermal scenes based on explicit thermodynamic modeling. We directly introduce explicit thermal physical variables into the Gaussian representation and derive the closed-form solution of the ordinary differential equation (ODE) at arbitrary time, avoiding the overhead of numerical integration and naturally adapting to unequal sampling and disordered timestamps. By combining explicit thermodynamic modeling with the Gaussian rendering framework, the proposed method is close to static 3DGS in training and inference efficiency, while achieving significantly better performance than recent state-of-the-art baselines in dynamic thermal reconstruction tasks. In addition, we also constructed an RHD dataset that covers typical processes such as warming, cooling, heating and heat transfer under controlled conditions, and provides millisecond timestamps and pixel-aligned RGB-IR data, thus laying the foundation for quantitative research on dynamic thermal scenes. Our contributions are summarized as follows:

- We propose a dynamic thermal scene reconstruction method based on explicit thermodynamic physics. We construct a reliable thermodynamic model using equivalent heat capacity, heat exchange coefficients, and heat source excitations. We derive closed-form solutions to ODEs at arbitrary time, enabling efficient rendering under unequal sampling and out-of-order sampling.
- We introduce the Rapid Heat Dynamics (RHD) dataset, a new dataset designed specifically for rapidly changing thermal dynamics. It covers the thermodynamic processes of warming, cooling, heating, and heat transfer, along with millisecond-accurate timestamps and pixel-aligned RGB data.
- Experimental results demonstrate that the proposed method reliably reconstructs rapidly changing thermal dynamics with comparable training and rendering efficiency to static 3DGS, and outperforms existing methods across various metrics.

## 2 RELATED WORK

### 2.1 3D THERMAL SCENE RECONSTRUCTION

3D thermal scene reconstruction aims to recover both scene geometry and thermal radiation, supporting applications such as energy efficiency monitoring, industrial inspection, and medical diagnosis (He et al. (2021); Glowacz (2021); Lahiri et al. (2012); Zhou et al. (2021)). Early methods adopted a two-stage strategy: reconstruct 3D structure from RGB images and then map thermal images (Dlesk et al. (2022); Jia et al. (2017); Zhao et al. (2017)). While feasible, these methods underuse thermal information and are restricted to static settings.

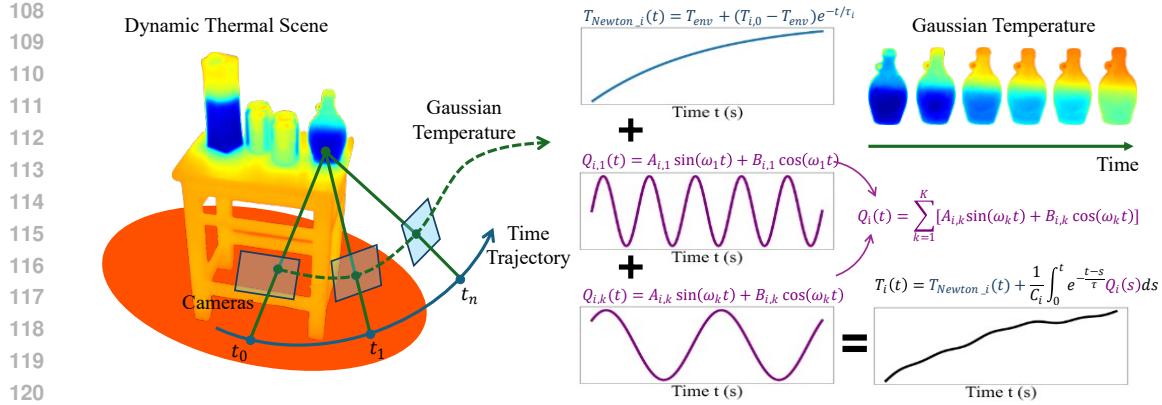


Figure 2: **Method Overview.** ETGS directly incorporates thermal physics modeling into the explicit Gaussian scene representation. The temperature of each Gaussian consists of two components: an exponential term ( $T_{Newton}$ ) that follows Newton’s law of cooling, describing its equilibrium tendency with the ambient temperature; and a heat source excitation term ( $Q$ ) expanded by a set of harmonic functions, representing periodic or complex external energy inputs. These two components combine to form a temperature evolution model.

Recent works explore neural implicit models. NeRF (Mildenhall et al. (2021)) has been extended to thermal imaging, e.g., ThermoNeRF (Hassan et al. (2024)) and Thermal-NeRF (Ye et al. (2024)), which learn 3D thermal distributions and synthesize novel views. However, implicit NeRFs suffer from unstable geometry and high training/rendering cost, limiting scalability to large or dynamic scenes.

Different from NeRF, 3D Gaussian Splatting (3DGS) (Kerbl et al. (2023)) adopts explicit Gaussians for efficient rendering and editability. Extensions such as Thermal3D-GS (Chen et al. (2024)) and TGA-GS (Zou et al. (2025)) apply 3DGS to thermal modeling, achieving faster rendering but still constrained to static scenes, thus failing to capture temporal temperature evolution.

## 2.2 DYNAMIC RECONSTRUCTION AND THERMODYNAMICS MODELING

To model dynamics, 4DGS (Wu et al. (2024)) extends Gaussians with deformation fields to capture geometry and appearance changes; follow-ups further improve efficiency (Wan et al. (2024); Lee et al. (2024); Zhang et al. (2025); Li et al. (2024c)). Yet, these methods address only appearance, not thermal physical properties, and are unsuitable for thermal dynamic scenes. ThermalGS (Liu et al. (2025)) introduces time embeddings for dynamic thermal modeling, but remains data-driven and prone to physical inconsistency. NTR-Gaussian (Yang et al. (2025)) incorporates thermodynamic parameters via neural prediction and numerical integration, improving physical fidelity but at high computational cost due to its implicit representation.

In contrast, our method directly embeds thermophysical parameters into Gaussian primitives and derives closed-form, differentiable time solutions. This design avoids numerical integration, maintains efficiency comparable to static 3DGS, and yields physically consistent dynamic thermal reconstructions.

## 3 METHOD

We present Explicit Thermal Gaussian Splatting (ETGS), which embeds thermophysical modeling into an explicit Gaussian representation. Each Gaussian carries geometry and a time-evolving temperature state governed by two components (Fig. 2): an exponential term from Newton’s cooling toward ambient equilibrium, and a heat source excitation expanded on harmonic bases to capture periodic/complex inputs. Their combination yields a closed-form, differentiable temperature evolution that can be evaluated at any timestamp—without numerical integration or implicit neural regression—thereby naturally handling unequal and out-of-order sampling. This design attains train-

162 ing/rendering efficiency close to static 3DGS while accurately reconstructing fast thermal dynamics,  
 163 striking a balance among accuracy, efficiency, and physical plausibility.  
 164

### 165 3.1 THERMAL GAUSSIAN FIELD

167 In 3DGS, a scene is an explicit Gaussian field of millions of parameterized Gaussians  $G_i$ . Its basic  
 168 form is:

$$169 \quad G_i = \{\mu_i, \Sigma_i, R_i, \alpha_i, f_i\}, \quad (1)$$

170 where  $\mu_i \in \mathbb{R}^3$  represents the center position of Gaussians,  $\Sigma_i$  represents the covariance matrix,  $R_i$   
 171 represents the rotation matrix,  $\alpha_i$  represents the opacity, and  $f_i$  represents the characteristic coeffi-  
 172 cient related to color or radiation (represented in the form of spherical harmonic function expansion).  
 173 This explicit representation makes 3DGS efficient and flexible in geometry and appearance model-  
 174 ing. However, for thermal scenes, the various properties of the Gaussians need to be redefined. As  
 175 the determining factor of thermal radiation, temperature is necessary to be explicitly represented  
 176 in modeling. To this end, we expand the Gaussians  $G_i$  into thermal Gaussians  $\tilde{G}_i$  and remove the  
 177 optical properties (spherical harmonic color) to focus on thermal reconstruction. The definition of  
 178 thermal Gaussians is:

$$179 \quad \tilde{G}_i = \{\mu_i, \Sigma_i, R_i, \alpha_i, C_i, h_i, Q_i(t), T_i(t)\}. \quad (2)$$

180 New thermal properties include: equivalent heat capacity  $C_i$ , which characterizes the inertia of the  
 181 Gaussians to temperature changes and determines how quickly the temperature responds to external  
 182 stimuli; heat transfer coefficient  $h_i$ , which describes the rate of heat exchange between the Gaus-  
 183 sian body and the environment; heat source excitation  $Q_i(t)$ , which represents the energy input to  
 184 the thermal Gaussians over time and is expanded using a set of Fourier basis functions to char-  
 185 acterize complex or periodic thermal processes; and temperature state  $T_i(t)$ , which represents the  
 186 temperature of the thermal Gaussians at time  $t$  and is analytically solved by a thermodynamic evo-  
 187 lution model (see Section 3.2). This design preserves the explicit controllability of the scene and  
 188 lays the foundation for physically consistent and efficient rendering of subsequent dynamic thermal  
 189 processes.

### 190 3.2 THERMODYNAMIC EVOLUTION

192 This section details the thermodynamic evolution of thermal Gaussians, including the first-order  
 193 linear differential equations and their analytical solutions, derived from Newtonian heat exchange  
 194 and heat source excitation, and the closed-form solution obtained after expanding the heat source  
 195  $Q_i(t)$  over a global frequency grid using a harmonic basis.

196 **Continuous ODE model.** For the  $i$ -th thermal Gaussian, assume the environment temperature is a  
 197 constant  $T_{env}$ , the equivalent heat capacity  $C_i > 0$ , the convection/radiation equivalent heat transfer  
 198 coefficient  $h_i \geq 0$ , and the heat source is  $Q_i(t)$ . Based on the energy conservation principle, the  
 199 first-order linear ODE is obtained:

$$201 \quad C_i \frac{dT_i(t)}{dt} = -h_i (T_i(t) - T_{env}) + Q_i(t). \quad (3)$$

203 Defining the time constant  $\tau_i = \frac{C_i}{h_i}$ , Eq. 3 can be rewritten as:

$$205 \quad \frac{dT_i(t)}{dt} = -\frac{1}{\tau_i} (T_i(t) - T_{env}) + \frac{1}{C_i} Q_i(t). \quad (4)$$

207 **Analytical solution of the ODE.** Applying the integrating factor method to Eq. 4 (see Appendix A  
 208 for the detailed derivation) and denoting the initial condition by  $T_i(0) = T_{i,0}$ , we obtain:

$$210 \quad T_i(t) = T_{env} + (T_{i,0} - T_{env}) e^{-\frac{t}{\tau_i}} + \frac{1}{C_i} \int_0^t e^{-\frac{t-s}{\tau_i}} Q_i(s) ds. \quad (5)$$

212 **Harmonic expansion of the heat source.** We expand the heat source on a globally shared frequency  
 213 grid:

$$214 \quad Q_i(t) = \sum_{k=1}^K A_{i,k} \sin(\omega_k t) + B_{i,k} \cos(\omega_k t), \quad (6)$$

216 where  $\{\omega_k\}_{k=1}^K$  are sampled from a log-uniform frequency grid:  
 217

$$218 \quad 219 \quad \omega_k = \omega_{\min} \left( \frac{\omega_{\max}}{\omega_{\min}} \right)^{\frac{k-1}{K-1}}, \quad k = 1, \dots, K. \quad (7)$$

221 The bounds  $[\omega_{\min}, \omega_{\max}]$  and the number of frequencies  $K$  are jointly determined by the sampling  
 222 geometry (observation duration  $T_{\text{span}}$ , minimum sampling interval  $dt_{\min}$ ) and the thermodynamic  
 223 priors ( $\tau_{\min}$ , steady-state gain threshold  $\alpha$ ) (see Appendix B for construction details).

224 **Closed-form solution of the ODE.** Substituting Eq. 6 into Eq. 5 and evaluating the kernel integrals  
 225 term by term (see Appendix A for the detailed derivation), we obtain a closed-form expression for  
 226 the temperature of each thermal Gaussian at arbitrary time  $t$ :

$$228 \quad T_i(t) = T_{\text{env}} + (T_{i,0} - T_{\text{env}}) e^{-t/\tau_i} \\ 229 \quad + \sum_{k=1}^K \frac{\tau_i}{C_i(1 + (\omega_k \tau_i)^2)} \left\{ A_{i,k} [\sin(\omega_k t) - \omega_k \tau_i \cos(\omega_k t) + \omega_k \tau_i e^{-t/\tau_i}] \right. \\ 230 \quad \left. + B_{i,k} [\cos(\omega_k t) + \omega_k \tau_i \sin(\omega_k t) - e^{-t/\tau_i}] \right\}. \quad (8)$$

### 235 3.3 DYNAMIC THERMAL RENDERING

236 During dynamic thermal rendering, temperatures need be mapped to colors. We adopt a linear  
 237 model for this mapping. Specifically, we linearly normalize temperatures to grayscale using the  
 238 temperature bounds measured during acquisition. Let the scene’s temperature range be  $[T_{\min}, T_{\max}]$   
 239 and the temperature of the  $i$ -th thermal Gaussian at time  $t$  be  $T_i(t)$ . The corresponding grayscale  
 240 intensity is:

$$242 \quad 243 \quad I_i(t) = \text{clip} \left( \frac{T_i(t) - T_{\min}}{T_{\max} - T_{\min}}, 0, 1 \right), \quad (9)$$

244 where  $\text{clip}(\cdot, 0, 1)$  truncates values to  $[0, 1]$ . During training, we use continuous intensity ( $[0, 1]$   
 245 interval) to participate in differentiable loss, and then map it to pseudo color during visualization to  
 246 enhance the visual effect.

247 For rendering, we perform standard alpha compositing along each ray while keeping the usual trans-  
 248mittance  $Tr_i$  and opacity  $\alpha_i$ , but use  $I_i(t)$  instead of the SH color term of 3DGS. The dynamic  
 249 thermal rendering follows the equation below:

$$252 \quad 253 \quad C = \sum_{i=1}^N Tr_i \alpha_i I_i(t). \quad (10)$$

### 255 3.4 TRAINING AND OPTIMIZATION

256 Our overall training and optimization framework follows the traditional 3DGS optimization process,  
 257 using the difference between the rendered image and the ground-truth image to drive gradient back-  
 258 propagation, thereby continuously updating the geometric and thermal properties of the Gaussians.  
 259 Based on the original 3DGS loss function, we introduce an additional regularization constraint based  
 260 on the characteristics of dynamic thermal scenarios to stabilize the learning of the heat source har-  
 261 monic parameters  $A_{i,k}$  and  $B_{i,k}$  to prevent overfitting and oscillation. The final total loss function  
 262 is:

$$264 \quad 265 \quad \mathcal{L}_{\text{total}} = (1 - \lambda) \mathcal{L}_1 + \lambda \mathcal{L}_{D-SSIM} + \lambda_{\text{reg}} \sum_{i,k} (A_{i,k}^2 + B_{i,k}^2), \quad (11)$$

266 Where  $\mathcal{L}_1$  and  $\mathcal{L}_{D-SSIM}$  are widely used in Nerf/3DGS-style reconstruction (Kerbl et al. (2023)),  
 267 their specific definitions can be found in Appendix E. This loss design allows our method to preserve  
 268 rendering accuracy while mitigating overfitting in thermal dynamics modeling, thereby ensuring  
 269 robust convergence even in complex dynamic scenes.

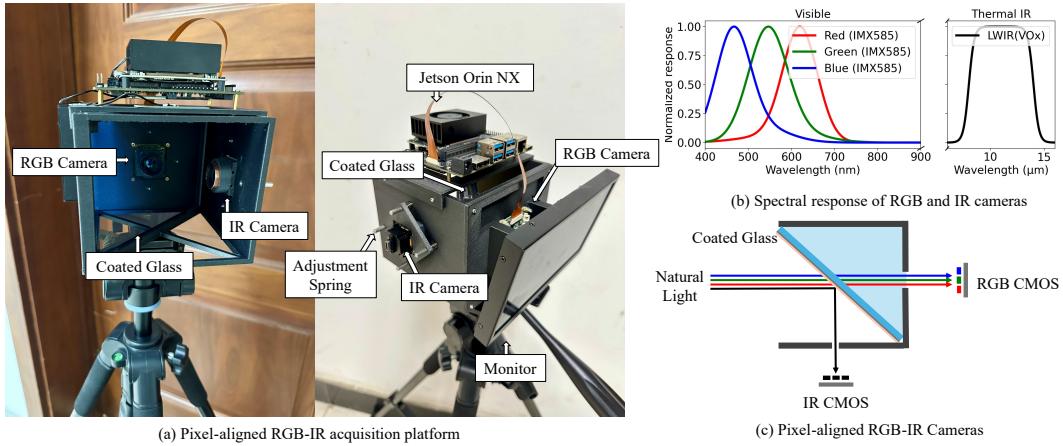


Figure 3: **Pixel-aligned RGB-IR acquisition platform.** (a) All devices are mounted on a rigid frame to ensure stability. (b) Response bands of the RGB and IR cameras: The RGB camera uses the IMX585 chip (response band 300-800nm), and the IR camera uses an uncooled vanadium oxide (VOx) microbolometer sensor (response band 8-14 $\mu$ m). (c) Optical principle of pixel-level alignment: A piece of coated glass (coating materials: zinc sulfide, silver) is mounted within the black frame. Visible light passes through the glass and enters the front RGB CMOS, while infrared light is reflected by the glass and reaches the side IR CMOS. The two imaging paths share the same incident light beam, which is split into different wavelengths at the beam splitter. Pixel-level alignment is achieved using a combination of fine-tuning springs and disposable camera calibration.

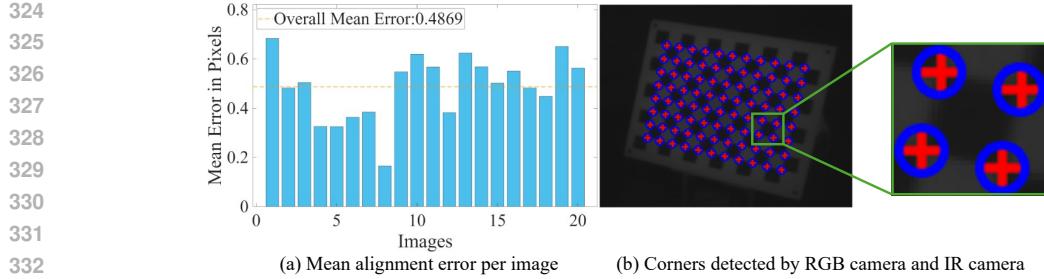
## 4 RHD: RAPID HEAT DYNAMICS DATASET

### 4.1 PIXEL-ALIGNED RGB-IR ACQUISITION PLATFORM.

To efficiently and economically acquire dynamic thermal scene data, we designed and constructed a pixel-level aligned RGB-IR acquisition platform, as shown in Fig. 3. The core optomechanical structure consists of a piece of selectively coated glass (coating materials: zinc sulfide, silver) positioned at a 45-degree angle. Visible light is transmitted to the front-facing RGB camera, while infrared light is reflected to the side-facing infrared camera, achieving coaxial imaging and zero-baseline beam splitting. The two images are time-stamped and synchronously captured by the Jetson Orin NX. For calibration and alignment, we first perform intrinsic distortion correction on each camera. We then estimate pixel-level alignment error on a common checkboard.

### 4.2 DATASET CONSTRUCTION.

We introduce a new dataset called Rapid Heat Dynamics (RHD), acquired using the pixel-aligned RGB-IR acquisition platform described in section 4.1. The basic specifications of the RHD dataset include: 512 $\times$ 410 resolution, 10 dynamic thermal scenes, and a total of 2410 views (4820 RGB and Thermal images). The RHD dataset covers dynamic thermal scenes with rapid temperature changes associated with typical thermodynamic processes, such as cooling, warming, heating, and heat transfer; covers a variety of materials, including metals, fabrics, and organic materials; and covers temperatures ranging from low to high temperatures (-1.0°C to 101.0°C). It also includes millisecond-accurate timestamps. For each scene, we provide pixel-level aligned RGB, Thermal (original), and Thermal (pseudo) images, as well as scene metadata (number of views, temperature range, time range, and ambient temperature), to facilitate both algorithm use and human interpretation. A detailed description of each scene is included in Appendix C. RHD focuses on both cross-modal geometric consistency and thermal dynamic richness, providing a high-quality benchmark for subsequent multimodal research, dynamic thermal scene rendering, and physical prior learning.



378 Table 1: Quantitative evaluation of our method compared to previous work.  
379

Metric	Method	Cooling Checkboard	Warming Bottles	Cooling Dumbbells	Cooling Bench	Cooling Ebike	Heat Transfer	Warming Peaches	Heating Workpieces	Warming Cups	Warming Workpieces	Avg.
PSNR↑	3DGS	36.27	32.37	29.78	29.74	34.66	39.47	30.48	34.47	24.43	29.94	32.16
	Mip-Splatting	36.48	32.44	28.78	29.13	35.16	37.45	32.74	30.06	26.18	26.67	31.51
	Thermal3D-GS	39.47	33.61	32.89	32.11	38.77	39.83	34.12	34.93	30.96	30.06	34.68
	4DGS	33.58	30.52	34.50	35.61	33.99	39.75	34.64	30.36	34.15	32.34	33.94
	NTR-Gaussian	41.82	34.63	35.18	35.56	38.09	38.99	32.44	33.65	28.18	31.01	34.96
	Ours	<b>44.73</b>	<b>40.23</b>	<b>37.63</b>	<b>38.25</b>	<b>41.67</b>	<b>42.27</b>	<b>41.78</b>	<b>36.70</b>	<b>39.41</b>	<b>44.16</b>	<b>40.68</b>
SSIM↑	3DGS	0.978	0.988	0.968	0.960	0.980	0.990	0.987	0.988	0.968	0.974	0.978
	Mip-Splatting	0.974	0.988	0.961	0.958	0.980	0.986	0.988	0.979	0.972	0.969	0.976
	Thermal3D-GS	0.983	0.989	0.981	0.970	0.983	0.990	0.990	0.989	0.978	0.976	0.983
	4DGS	0.963	0.979	0.976	0.967	0.962	0.986	0.984	0.960	0.975	0.970	0.972
	NTR-Gaussian	0.986	0.987	0.980	0.974	0.974	0.987	0.984	0.982	0.974	0.978	0.981
	Ours	<b>0.987</b>	<b>0.992</b>	<b>0.989</b>	<b>0.983</b>	<b>0.985</b>	<b>0.991</b>	<b>0.994</b>	<b>0.990</b>	<b>0.989</b>	<b>0.994</b>	<b>0.989</b>
LPIPS↓	3DGS	0.066	0.087	0.072	0.075	0.031	0.057	0.091	0.031	0.131	0.137	0.078
	Mip-Splatting	0.068	0.080	0.093	0.078	0.031	0.075	0.073	0.043	0.126	0.178	0.085
	Thermal3D-GS	0.060	0.095	0.050	0.063	0.029	0.062	0.092	0.030	0.110	0.128	0.072
	4DGS	0.101	0.075	0.047	0.076	0.065	0.056	0.064	0.073	0.105	0.099	0.076
	NTR-Gaussian	0.072	0.092	0.052	0.092	0.098	0.092	0.104	0.055	0.133	0.102	0.089
	Ours	<b>0.054</b>	<b>0.072</b>	<b>0.030</b>	<b>0.050</b>	<b>0.028</b>	<b>0.054</b>	<b>0.060</b>	<b>0.028</b>	<b>0.085</b>	<b>0.042</b>	<b>0.050</b>

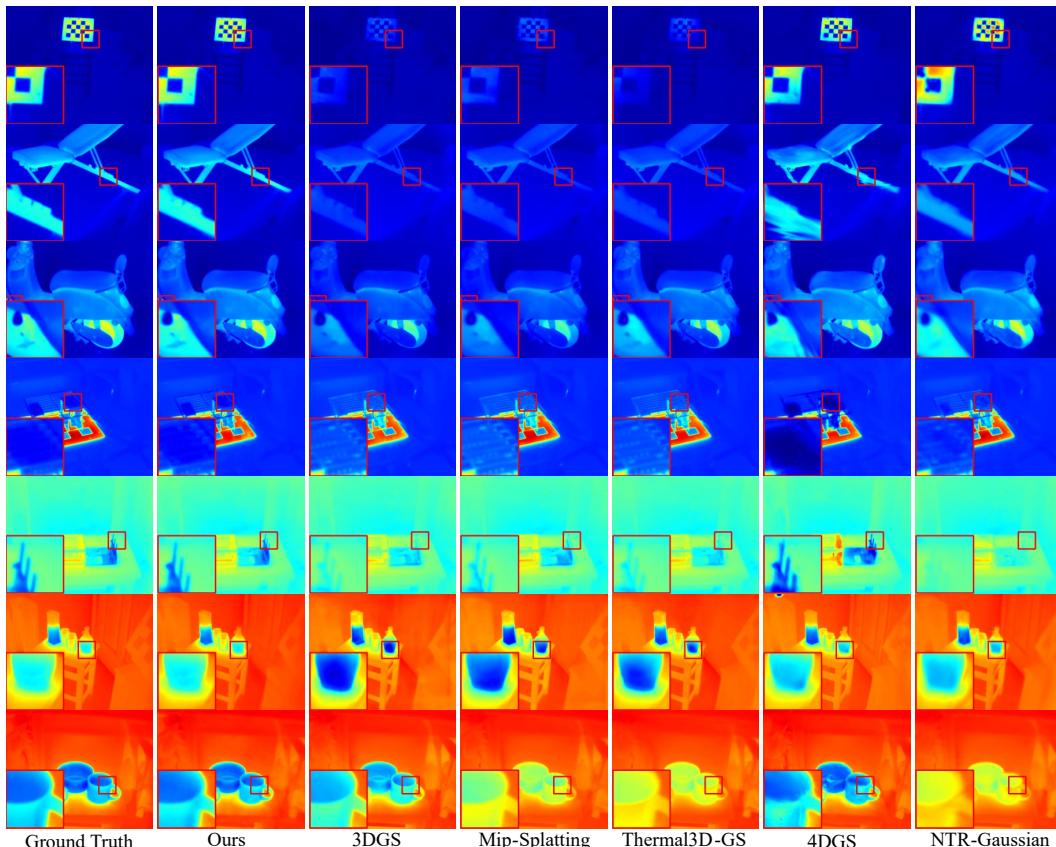
423 Figure 5: **Comparisons of ours to previous methods.** The scenes are, from the top down: Cooling  
424 Checkboard, Cooling Bench, Cooling Ebike, Heating Workpieces, Heat Transfer, Warming Bottles,  
425 Warming Cups from the RHD dataset.  
426  
427428 Fig. 5 shows a visual comparison of different methods in typical scenes. It can be observed that the  
429 static scene reconstruction methods (3DGS, Mip-Splatting and Thermal3D-GS) exhibit significant  
430 temperature deviations. This is because these static methods can only learn the average temperature  
431 of the scene and cannot obtain accurate temperature values. Dynamic scene reconstruction methods  
432 (4DGS and NTR-Gaussian) exhibit varying degrees of artifacts around the edges of some objects.

Table 2: Comparisons of training and rendering efficiency of our method with previous methods.

Method	Mem (MB) $\downarrow$	Time (s) $\downarrow$	FPS $\uparrow$
3DGs	2429	166	557
Mip-Splatting	2749	207	589
Thermal3D-GS	3265	470	342
4DGs	2290	1159	278
NTR-Gaussian	4439	1469	68
Ours	2391	197	458

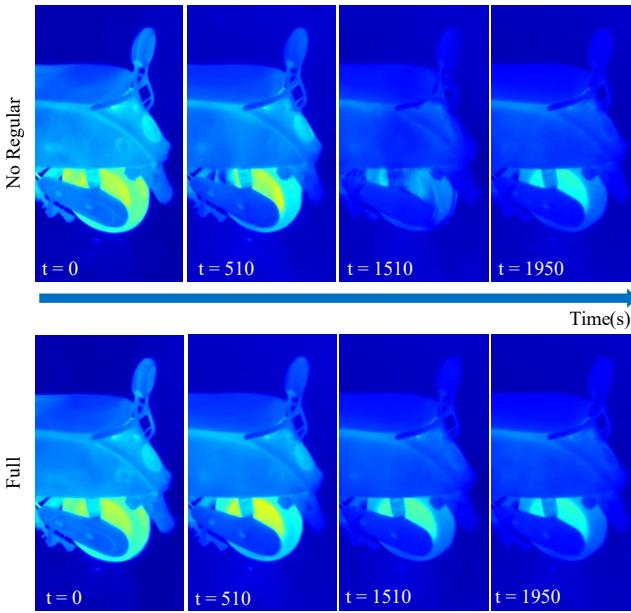


Figure 6: **Visualization of the ablation study.** Left: Ablation of the regularization term. Right: Ablation of the heat source excitation  $Q$ .

This is because their implicit modeling of time  $t$  makes it difficult to ensure temporal consistency. Our method's explicit modeling avoids this drawback, making the model more closely resemble the actual thermal distribution. We provide a detailed visual comparison of all scenes in Appendix F.

Table 4: Ablation Study. Effect of the number of frequencies  $K$  on reconstruction quality.

Metric	K=8	K=16	K=24	K=32	K=64
PSNR $\uparrow$	40.57	40.59	40.68	40.77	40.95
SSIM $\uparrow$	0.989	0.989	0.989	0.989	0.990
LPIPS $\downarrow$	0.050	0.051	0.050	0.051	0.051

Tab. 2 summarizes average gpu memory, training time, and rendering speed over all scenes. Our method trains in 197 s—close to 3DGS and Mip-Splatting and about an order of magnitude faster than 4DGS and NTR-Gaussian—and renders comparably to static baselines while surpassing implicit methods. This efficiency follows from combining explicit Gaussians with a closed-form temperature solution, avoiding repeated neural-field evaluations. **The Appendix D shows the complete calculation cost for ETGS, static baselines and dynamic baselines.**

## 5.4 ABLATION STUDY

We removed the heat source excitation  $Q$  and the regularization term to analyze their roles in the overall framework. Quantitative results are shown in Tab. 3, and the visualization results are shown in Fig. 6. We evaluated the impact of the number of frequencies  $K$  in the heat source excitation  $Q$  on the reconstruction results, and the results are shown in Tab. 4.

9

486    **Heat source excitation  $Q$ :** After removing the heat source term, the model can only rely on the  
 487    exponential decay of Newtonian heat transfer to describe temperature changes, resulting in a lack  
 488    of external driving force for scene evolution. In this case, the model often fails to capture complex  
 489    temperature variations or periodic fluctuations, and the learned temperature field exhibits underfitting.  
 490    As shown on the right side of Fig. 6, after removing  $Q$ , the model misses fine details and edges  
 491    in the scene.

492    **Regularization term:** The regularization term that we introduced constrains the parameters  $A_{i,k}$   
 493    and  $B_{i,k}$  of the frequency basis expansion to prevent them from uncontrolled amplification. After  
 494    removing this term, although the model still fits the overall trend, unphysical oscillations occur over  
 495    long time series. This causes the temperature evolution curve to deviate from the smooth physical  
 496    law, introducing noticeable fluctuation artifacts in the rendering (Fig. 6, left).

497    **Number of frequencies  $K$ :** As the number of frequencies  $K$  increases, performance improves  
 498    slightly, but the improvement quickly saturates. Even smaller  $K$  (e.g., 8-16) are sufficient to capture  
 499    the main frequencies, while  $K = 24$  strikes a good balance between accuracy and computational  
 500    cost. Therefore, we used this value in our main experiments. Increasing  $K$  above 32 yields very  
 501    slight improvements (PSNR < 0.2 dB, SSIM and LPIPS remain almost unchanged).

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## 504    6 DISCUSSION

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506    **Independent and Coupled Modeling of Gaussians.** Although ETGS models the thermodynamic  
 507    evolution of each Gaussian as an independent thermodynamic process, the rendered thermal field  
 508    is not independent between different Gaussians, but rather implicitly coupled. This naturally stems  
 509    from two mechanisms:(1) overlapping Gaussians. Adjacent Gaussians contribute to the same pixels  
 510    in the rendered image. Any temperature discontinuity between adjacent Gaussian bodies im-  
 511    mediately produces visible artifacts. (2) dense supervision. Infrared images provide hundreds of  
 512    thousands of constraints for each view. During optimization, the renderer forces Gaussians in lo-  
 513    cal regions to match the same pixel observations, which implicitly achieves smooth temperature  
 514    changes between adjacent Gaussians. Explicitly modeling heat conduction between Gaussians is  
 515    an interesting direction for future research. However, it introduces enormous computational and  
 516    implementation complexity: global coupling between hundreds of thousands of Gaussians, the in-  
 517    ability to obtain closed-form solutions, and significantly increased backpropagation costs. ETGS  
 518    focuses on lightweight, closed-form solutions that are physically accurate and capture the main ther-  
 519    modynamics we observe in controlled dynamic scenes, while also extending explicit inter-Gaussian  
 520    conduction as an important feature research direction.

521    **More complex thermodynamic models.** ETGS adopts a first-order linear heat-transfer model con-  
 522    sisting of Newtonian cooling and harmonic heat-source excitation. This formulation is intentionally  
 523    chosen because it admits a closed-form analytical solution, enables efficient optimization compara-  
 524    ble to static 3DGS. However, real-world thermodynamic processes may involve nonlinear effects  
 525    such as temperature-dependent conductivity, radiation coupling, multi-layer material interfaces, or  
 526    even phase transitions. These phenomena are difficult to incorporate into a closed-form solution, and  
 527    typically require solving nonlinear PDEs or numerical approximations, which would significantly  
 528    increase computational cost and compromise the efficiency advantage of explicit Gaussian splatting.  
 529    Extending ETGS to support nonlinear or higher-order thermal dynamics is an important direction.  
 530    Developing such models would allow ETGS to generalize to more complex thermal scenarios, in-  
 531    cluding outdoor environments, heterogeneous materials, or strong radiative interactions.

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## 534    7 CONCLUSION

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536    ETGS embeds thermophysics into explicit Gaussians and derives a closed-form temperature solution  
 537    for efficient, stable dynamic thermal reconstruction. We also release RHD with pixel-aligned RGB-  
 538    IR and ms-level timing. In future work, we plan to expand RHD to include moving heat sources and  
 539    more complex environments. Combining RGB with thermal supervision is also a very promising  
 direction to explore.

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## APPENDIX

We organize the appendix as follows:

- **Appendix A:** A complete derivation of the thermodynamic evolution of thermal Gaussians is given (starting from Newtonian heat transfer, obtaining an analytical solution via the integrating factor method, and deriving a closed-form differentiable expression at any time under a harmonic heat source).
- **Appendix B:** Detailed description of the construction principles and implementation of the frequency grid, the determination of upper and lower bounds for sampling and thermal prior constraints, logarithmic sampling and minimum frequency separation, and engineering treatment for robustness.
- **Appendix C:** Report on the scene and time statistics, temperature range, and metadata of the RHD dataset to support reproduction and benchmarking.
- **Appendix D:** The complete calculation cost for ETGS, static baselines and dynamic baselines.
- **Appendix E:** Details about the definitions of common loss functions.
- **Appendix F:** A complete visual comparison of each scene (including local zoom) is provided as an intuitive supplement to the quantitative indicators in the main paper.
- **Appendix G:** Outlook for the potential expansion of the method towards more complex thermal processes and multimodal fusion.
- **Appendix H:** The Use of Large Language Models (LLMs).

## A COMPLETE DERIVATION OF THERMODYNAMIC EVOLUTION

**Linearization and Integrating Factor:** Rewrite Eq. 3 into standard linear form:

$$\frac{dT_i}{dt} + \frac{1}{\tau_i} T_i = \frac{1}{\tau_i} T_{\text{env}} + \frac{1}{C_i} Q_i(t). \quad (13)$$

Let the integrating factor be  $\mu(t) = e^{t/\tau_i}$ . Multiplying both sides of the equation by  $\mu(t)$  and then differentiating, we obtain:

$$\frac{d}{dt} \left( e^{t/\tau_i} T_i(t) \right) = \frac{1}{\tau_i} T_{\text{env}} e^{t/\tau_i} + \frac{1}{C_i} Q_i(t) e^{t/\tau_i}. \quad (14)$$

Integrating over  $t \in [0, t]$  and denoting the initial condition  $T_i(0) = T_{i,0}$ , then obtain:

$$e^{t/\tau_i} T_i(t) - T_{i,0} = \frac{1}{\tau_i} T_{\text{env}} \int_0^t e^{s/\tau_i} ds + \frac{1}{C_i} \int_0^t e^{s/\tau_i} Q_i(s) ds, \quad (15)$$

where multiplying both sides by  $e^{-t/\tau_i}$  gives:

$$T_i(t) = T_{\text{env}} + (T_{i,0} - T_{\text{env}}) e^{-t/\tau_i} + \frac{1}{C_i} \int_0^t e^{-(t-s)/\tau_i} Q_i(s) ds. \quad (16)$$

Eq. 16 is the same as Eq. 5 in the main text.

**Heat source and kernel convolution:** Eq. 6 in the main text is given as:

$$Q_i(t) = \sum_{k=1}^K (A_{i,k} \sin(\omega_k t) + B_{i,k} \cos(\omega_k t)). \quad (17)$$

756 Two types of kernel convolutions need to be evaluated:  
 757

$$758 \quad 759 \quad I_{\sin}(\omega, t) = \int_0^t e^{-(t-s)/\tau_i} \sin(\omega s) ds, \quad (18)$$

$$760$$

$$761 \quad 762 \quad I_{\cos}(\omega, t) = \int_0^t e^{-(t-s)/\tau_i} \cos(\omega s) ds. \quad (19)$$

$$763$$

764 Let  $a = 1/\tau_i$  and  $b = \omega$ . Using the indefinite integral identities:  
 765

$$766 \quad 767 \quad \int e^{as} \sin(bs) ds = \frac{e^{as}}{a^2 + b^2} (a \sin bs - b \cos bs), \quad (20)$$

$$768$$

$$769 \quad 770 \quad \int e^{as} \cos(bs) ds = \frac{e^{as}}{a^2 + b^2} (a \cos bs + b \sin bs). \quad (21)$$

$$771$$

772 Using  $e^{-(t-s)/\tau_i} = e^{-t/\tau_i} e^{s/\tau_i}$  to simplify, we obtain the following definite integrals:  
 773

$$774 \quad 775 \quad I_{\sin}(\omega, t) = \frac{1}{a^2 + b^2} \left( \frac{1}{\tau_i} \sin \omega t - \omega \cos \omega t + \omega e^{-t/\tau_i} \right), \quad (22)$$

$$776$$

$$777 \quad 778 \quad I_{\cos}(\omega, t) = \frac{1}{a^2 + b^2} \left( \frac{1}{\tau_i} \cos \omega t + \omega \sin \omega t - \frac{1}{\tau_i} e^{-t/\tau_i} \right). \quad (23)$$

$$779$$

780 Substituting  $a = 1/\tau_i$  and using  $a^2 + b^2 = \frac{1+\omega^2\tau_i^2}{\tau_i^2}$  yields the compact forms:  
 781

$$782 \quad 783 \quad I_{\sin}(\omega, t) = \frac{\tau_i}{1 + \omega^2 \tau_i^2} \left( \sin \omega t - \omega \tau_i \cos \omega t + \omega \tau_i e^{-t/\tau_i} \right), \quad (24)$$

$$784$$

$$785 \quad 786 \quad I_{\cos}(\omega, t) = \frac{\tau_i}{1 + \omega^2 \tau_i^2} \left( \cos \omega t + \omega \tau_i \sin \omega t - e^{-t/\tau_i} \right). \quad (25)$$

$$787$$

788 Substituting Eq. 24 and Eq. 25 into the convolution term of Eq. 16, we obtain the final closed-form  
 789 solution:  
 790

$$791 \quad 792 \quad T_i(t) = T_{\text{env}} + (T_{i,0} - T_{\text{env}}) e^{-t/\tau_i}$$

$$793 \quad 794 \quad + \sum_{k=1}^K \frac{\tau_i}{C_i(1 + (\omega_k \tau_i)^2)} \left\{ A_{i,k} [\sin(\omega_k t) - \omega_k \tau_i \cos(\omega_k t) + \omega_k \tau_i e^{-t/\tau_i}] \right.$$

$$795 \quad 796 \quad \left. + B_{i,k} [\cos(\omega_k t) + \omega_k \tau_i \sin(\omega_k t) - e^{-t/\tau_i}] \right\}, \quad (26)$$

$$797$$

798 which corresponds to Eq. 8 in the main text.  
 799

## 800 B FREQUENCY BOUNDS: JOINT CONSTRAINTS FROM SAMPLING AND 801 THERMODYNAMICS

802 We expand the heat source  $Q_i(t)$  on a log-uniform frequency grid, whose passband  $[\omega_{\min}, \omega_{\max}]$   
 803 is determined jointly by the observation duration and sampling rate (signal-processing perspective)  
 804 and thermodynamic time-constant priors (physical perspective), balancing expressiveness and ro-  
 805 bustness. The initial number of frequencies is set to  $K = 24$ , and is finally adapted to the number  
 806 of deduplicated active frequencies, i.e.,  $K = |\{\omega_k\}|$ .  
 807

808 **Lowest frequency  $f_{\min}$ : Determined by the observation duration.** A finite observation duration  
 809  $T_{\text{span}}$  sets the minimum resolvable frequency grid  $\Delta f \approx 1/T_{\text{span}}$ . In other words, if a frequency

810 is lower than  $1/T_{span}$ , a full cycle cannot fit within the window and is indistinguishable from even  
 811 lower components. Hence we set:

$$813 \quad 814 \quad 815 \quad f_{min} = \max \left( \frac{1}{T_{span}}, 10^{-5} \right). \quad (27)$$

816 **Sampling upper bound  $f_{nyq}$ : Constrained by the Nyquist frequency (Shannon (2006)).** For  
 817 nonuniform timestamps with minimum interval  $dt_{min}$ , we define an effective maximum sampling  
 818 rate  $f_s^{max} \approx 1/dt_{min}$ , which yields a strict anti-aliasing bound:

$$820 \quad 821 \quad f_{nyq} = \frac{f_s^{max}}{2} \approx \frac{1}{2dt_{min}}. \quad (28)$$

822 **Thermodynamics upper bound  $f_{th}$ : Given by the first-order thermal inertia low-pass charac-  
 823 teristic.** Model each thermal Gaussian as a first-order linear thermal system with time constant  $\tau_i$ .  
 824 Its steady-state amplitude-frequency response to the angular frequency  $\omega$  is:

$$826 \quad 827 \quad 828 \quad |G(j\omega)| = \frac{1}{\sqrt{1 + (\omega\tau)^2}}, \quad (\tau > 0) \quad (29)$$

830 When the frequency increases, the system exhibits a low-pass characteristic and decays at  $1/\omega$ . We  
 831 require that the system can observe the heat source at the candidate frequency (the steady-state gain  
 832 is not less than the threshold  $\alpha \in (0, 1]$ , then

$$834 \quad |G(j\omega)| \geq \alpha, \quad (30)$$

835 equals

$$837 \quad 838 \quad 839 \quad \omega \leq \frac{1}{\tau} \sqrt{\frac{1}{\alpha^2} - 1}. \quad (31)$$

840 The corresponding upper limit of the frequency is

$$843 \quad 844 \quad f_{th} = \frac{1}{2\pi\tau_{min}} \sqrt{\max \left( \frac{1}{\alpha^2} - 1, 0 \right)}, \quad (32)$$

845 where we use the fastest thermal time constant prior  $\tau_{min}$  (i.e., the “minimum thermal inertia” of  
 846 the material/structure) to provide the most permissive physical upper bound. In the special case of  
 847 choosing the half-power point  $\alpha = 1/\sqrt{2}$ , the well-known cutoff frequency is obtained as  $f_c =$   
 848  $1/(2\pi\tau)$ . Moreover, the time constant  $\tau$  satisfies  $\tau = C/h$ , where  $C$  is the equivalent heat capacity  
 849 and  $h$  is the heat transfer coefficient in Newton’s law of cooling. The exponential decay solution in  
 850 the time domain and the first-order low-pass behavior in the frequency domain are two consistent  
 851 characterizations of the same thermal process Bergman (2011).

852 **Joint frequency constraints:** Combining the sampling and physical constraints, the heat source  
 853 frequency bounds are set to

$$855 \quad 856 \quad \omega_{min} = 2\pi f_{min}, \quad (33)$$

$$858 \quad 859 \quad \omega_{max} = 2\pi \max(\min(f_{nyq}, f_{th}), 1.2 f_{min}). \quad (34)$$

860 We first take  $\min(f_{nyq}, f_{th})$  to ensure both sampling safety and thermodynamic responsiveness.  
 861 Then compare this value against  $1.2 f_{min}$  to guarantee a nondegenerate bandwidth-i.e., even under  
 862 extremely sparse or short observation windows, the usable band remains at least one frequency-bin  
 863 wide. This prevents the collapse of the upper and lower bounds to the same point, which would  
 invalidate log-spaced sampling of the frequency grid.

864 C RHD: RAPID HEAT DYNAMICS DATASET  
865

866 As summarized in Tab. 5 and Tab. 6, we present our RHD dataset. Built with the pixel-aligned RGB-  
867 IR acquisition platform of Section 4.1, RHD has a resolution of 512×410, comprises 10 dynamic  
868 thermal scenes, and totals 2,410 viewpoints (4,820 images; one RGB and one thermal per view)  
869 with millisecond-accurate timestamps. RHD covers canonical thermodynamic processes-cooling,  
870 warming, heating, and heat transfer-and spans diverse materials including metals, fabrics, complex  
871 devices, and organic objects, with a temperature range of -1.0 °C to 101.0 °C. For each scene, we  
872 provide pixel-aligned RGB, Thermal (raw radiometric grayscale), Thermal (pseudocolor), and scene  
873 metadata (number of views, temperature range, time span, ambient temperature). With its rich cross-  
874 modal and thermal dynamics content, RHD serves as a high-quality benchmark for multimodal 3D  
875 reconstruction, dynamic thermal scene rendering, and learning with physical priors.

876  
877 Table 5: Each scene of the Rapid Heat Dynamics Dataset.  
878

879 Scene	880 RGB	881 Thermal(original)	882 Thermal(pseudo)	883 Views	884 Temp. Range (°C)	885 Time Range (s)	886 Env. Temp (°C)
887 Cooling Checkboard	888	889	890	891 238	892 20.0 72.0	893 2145.488	894 26.5
895 Cooling Dumbbells	896	897	898	899 210	900 28.0 46.0	901 2115.472	902 32.3
903 Cooling Bench	904	905	906	907 233	908 31.0 56.0	909 1689.206	910 32.9
911 Cooling Ebike	912	913	914	915 316	916 26.0 60.0	917 1954.054	918 30.2
919 Heat Transfer	920	921	922	923 256	924 21.0 41.0	925 1730.072	926 26.4
927 Heating Workpieces	928	929	930	931 232	932 12.0 101.0	933 1249.084	934 26.9
935 Warming Bottles	936	937	938	939 209	940 5.0 31.0	941 3988.082	942 26.6

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Table 6: Each scene of the Rapid Heat Dynamics dataset.

Scene	RGB	Thermal(original)	Thermal(pseudo)	Views	Temp. Range (°C)	Time Range (s)	Env. Temp (°C)
Warming Cups				224	-1.0 30.0	1678.116	26.9
Warming Peaches				254	2.0 30.0	2284.465	26.9
Warming Workpieces				238	3.0-32.0	1554.317	26.9

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Table 7: Complete calculation cost.

Metric	Method	Cooling Checkboard	Warming Bottles	Cooling Dumbbells	Cooling Bench	Cooling Ebike	Heat Transfer	Warming Peaches	Heating Workpieces	Warming Cups	Warming Workpieces	Avg.
Mem↓(MB)	3DGS	2395	2313	2597	2324	2674	2372	2350	2340	2404	2522	2429.10
	Mip-Splatting	3389	2215	3417	2384	2694	2908	2154	2384	2810	3132	2748.70
	Thermal3D-GS	3207	3146	3497	3242	3544	3118	2970	3522	3154	3248	3264.80
	4DGS	2571	1953	3043	1846	3621	1846	1890	2444	1794	1894	2290.20
	NTR-Gaussian	4544	4344	4144	4130	4286	4598	4490	4656	4454	4740	4438.60
Train↓(s)	ours	2287	2311	2367	2336	2700	2408	2406	2348	2384	2362	2390.90
	3DGS	145	177	195	141	166	173	149	160	175	176	165.70
	Mip-Splatting	215	205	225	187	207	209	193	215	198	217	207.10
	Thermal3D-GS	441	438	527	450	454	431	415	522	433	593	470.40
	4DGS	791	847	905	1603	1521	1637	1621	1054	798	814	1159.10
FPS↑	NTR-Gaussian	1741	1298	1068	1025	1984	1486	1374	1644	1358	1711	1468.90
	ours	129	203	279	185	188	188	205	208	186	199	197.00
	3DGS	537	550	422	584	562	602	613	544	581	575	557.00
	Mip-Splatting	630	648	132	606	658	633	643	613	671	652	588.60
	Thermal3D-GS	344	342	305	351	351	356	352	312	349	357	341.90
FPS↑	4DGS	284	274	279	284	279	278	283	251	287	281	278.00
	NTR-Gaussian	62	67	77	83	85	60	65	58	65	55	67.70
	ours	486	442	396	458	449	487	477	421	478	489	458.30

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D COMPLETE CALCULATION COST

961 Tab. 7 provides a complete comparison of training memory, training time, and rendering FPS across  
962 all ten scenes in RHD. ETGS achieves a highly competitive computational profile. In terms of training  
963 memory, ETGS requires 2390 MB on average, which is significantly lower than Thermal3D-GS  
964 (3265 MB), and dramatically lower than NTR-Gaussian (4439 MB). This demonstrates that the ex-  
965 plicit thermodynamic modeling introduces minimal memory overhead compared with static 3DGS,  
966 and is far more memory-efficient than dynamic baselines. In terms of training time, ETGS trains in  
967 197 seconds on average, which is much faster than all dynamic baselines (e.g., 4DGS: 1159 s, NTR-  
968 Gaussian: 1469 s), and slightly slower than static 3DGS (166 s). In terms of rendering speed, ETGS  
969 achieves 458 FPS, which is very close to 3DGS (557 FPS), and far above all dynamic Nerf/3DGS  
970 methods. This confirms that the closed-form temperature solution introduces no runtime bottle-  
971 neck, preserving the hallmark real-time rendering performance of 3DGS. Overall, ETGS maintains  
972 the computational efficiency of static 3D Gaussian Splatting while providing physically grounded  
973 dynamic thermal modeling.

972 E DEFINITIONS OF COMMON LOSS FUNCTIONS  
973974 Photometric Loss  $\mathcal{L}_1$ :

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976 
$$\mathcal{L}_1 = \|I_{\text{rendered}} - I_{\text{gt}}\|_1, \quad (35)$$

977 Where  $I_{\text{rendered}}$  represents the rendered image and  $I_{\text{gt}}$  represents the ground truth image. This  
978 pixel-wise  $\mathcal{L}_1$  loss is the standard reconstruction objective used in NeRF/3DGS-style optimization.

980 D-SSIM (Structural Dissimilarity) Loss:

981  
982 
$$\mathcal{L}_{\text{D-SSIM}} = \frac{1 - \text{SSIM}(I_{\text{rendered}}, I_{\text{gt}})}{2}, \quad (36)$$

983 Where  $\text{SSIM}$  represents Structural Similarity Index, which is used to measure the similarity be-  
984 tween two images in terms of structure, brightness, and contrast (Wang et al. (2004)).985  
986 F COMPLETE VISUAL COMPARISON RESULTS  
987988 Fig. 7 shows a visual comparison of all scenes from the RHD dataset. This overall comparison  
989 demonstrates the advantages of ETGS in detail fidelity and temporal consistency. 3DGS, Mip-  
990 Splatting, and Thermal3D-GS are relatively stable in terms of structural preservation, but they lack  
991 the detail and dynamic consistency of temperature gradients, making them unable to reproduce time-  
992 varying thermal processes. 4DGS and NTR-Gaussian attempt to model the time dimension, but their  
993 implicit modeling makes it difficult to ensure temporal consistency. Our method maintains sharper  
994 edges, more reasonable temperature gradients, and dynamic evolution consistent with the actual  
995 thermal distribution in all scenes.996  
1000 G FUTURE WORK  
10011002 While ETGS has made significant progress in dynamic thermal scene reconstruction, several areas  
1003 remain worth exploring.1004 Extending the RHD dataset with more complex thermal processes. While RHD provides  
1005 millisecond-level RGB-IR observations for a wide variety of thermal processes, future versions of  
1006 the dataset will incorporate moving or time-varying heat sources, stronger environmental distur-  
1007 bances, and more diverse materials. These additions will enable evaluating thermal reconstruction  
1008 methods under significantly more challenging real-world conditions.1009  
1011 Coupled RGB-IR modeling. A natural extension of ETGS is to integrate a physically in-  
1012 formed RGB renderer to jointly model appearance and temperature. Such a coupled model could  
1013 simulate: temperature-dependent optical effects, including glowing surfaces or emissive materials  
1014 at high temperatures; appearance changes due to radiative transfer, refraction, or thermal reflection,  
1015 allowing the RGB channel to reflect thermal variations more faithfully. Integrating such cross-modal  
1016 interactions would enable richer multimodal reconstruction and enhance applications in monitoring,  
1017 inspection, and simulation.1018 Joint modeling of geometry and temperature evolution. The current ETGS formulation  
1019 focuses on dynamic thermal processes under a fixed geometry. A challenging but exciting direction  
1020 is to extend ETGS to simultaneously model geometric deformation and temperature evolution,  
1021 potentially by combining our thermodynamic formulation with deformation fields, dynamic  
1022 Gaussians, or canonical-space warping techniques. This would allow ETGS to handle scenes where  
1023 both structure and thermal state change over time.1024  
1025 Developing a dynamic thermal reconstruction framework that is both interpretable and gener-  
1026 alizable will be a key area of future research.

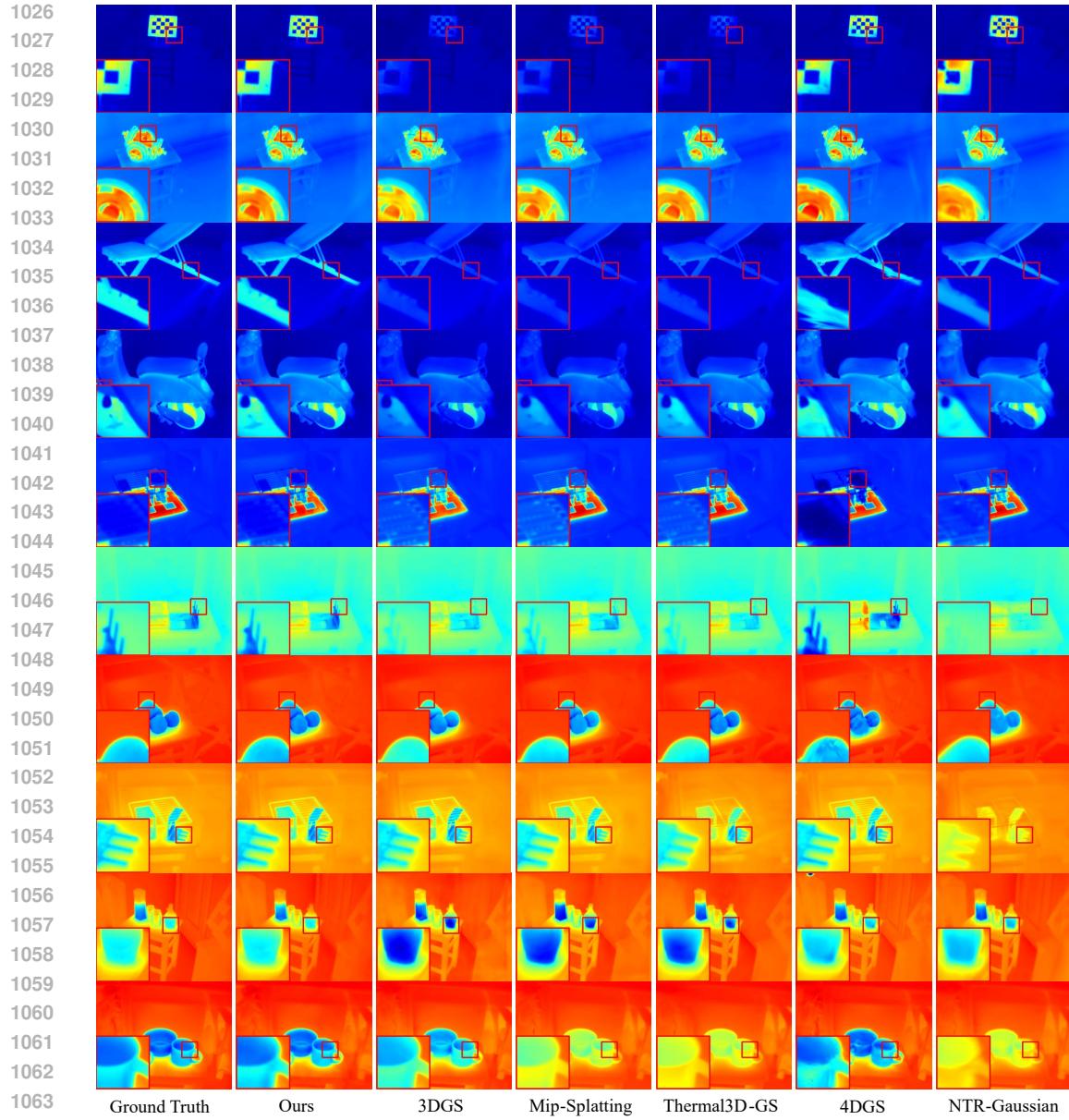


Figure 7: **Comparisons of ours to previous methods on all scenes.** The scenes are, from the top down: Cooling Checkboard, Cooling Dumbbells, Cooling Bench, Cooling Ebike, Heating Workpieces, Heat Transfer, Warming Peaches, Warming Workpieces, Warming Bottles, Warming Cups from the RHD dataset.

## H THE USE OF LARGE LANGUAGE MODELS (LLMs)

In preparing this manuscript, we employed Large Language Models (LLMs) for language polishing and stylistic refinement, with the goal of improving readability, clarity, and presentation quality.